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Analysis of chip formation mechanisms and modelling of slabber process

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Guillaume Fromentin

Abstract During the primary transformation in wood industry, logs are faced with conical rough milling cutters commonly named slabber or canter heads. Chips produced consist of raw materials for pulp paper and particleboard industries. The process efficiency of these industries partly comes from particle size distribution. However, chips formation is greatly dependent on milling conditions and material variability.

Thus, this study aims at better understanding and predicting chips production in wood milling. The different mechanisms of their formation are studied through orthogonal cutting experiments at high cutting speed for beech and Douglas fir. Within these conditions, ejection of free water inside wood can be observed during fragmentation, particularly on beech. As previously seen in quasi-static experiments, chip thickness is proportional to the nominal cut thickness. Moreover, the grain orientation has a great influence on the cutting mechanisms, so as the nominal cut and the growth circles widths. This chip fragmentation study finally allows the improvement of the cutting conditions in rough milling.

In order to optimize machine design as well as cutting geometry, a geometrical model of a generic slabber head is developed. This model allows the study of the effective cutting kinematics, the log-cutting edges interactions and the effective wood grain direction during cutting. This paper describes the great influence of the carriage position on cutting conditions. The results obtained here can be directly used by milling machine manufacturers.

Keywords geometrical modelling · chip fragmentation · slabber · wood

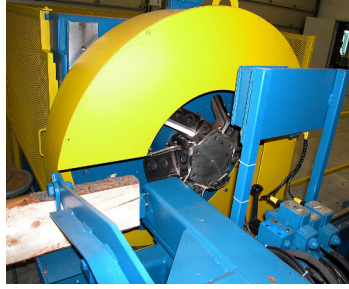
1 Introduction

The aim of sawmills consists in valorizing the maximum volume of wood from logs during the primary transformation. After cross cutting and debarking, logs are squared by rough milling operations with slabber heads i.e. conical milling cutters. During this operation, up to 30% of the logs initial volume is transformed into chips. These chips are mainly used as raw materials in pulp industries for paper and fiberboards. Defibration process is a key step for pulp industry. It can be mechanical, chemical or both. The chip size distribution, which affects directly the efficiency of the defibration process (Felber and Lackner, 2005), must be controlled. This distribution is greatly dependent on milling conditions and on variability in material. This paper aims at providing a better understanding of chip fragmentation so as to improve cutting conditions and machine design.

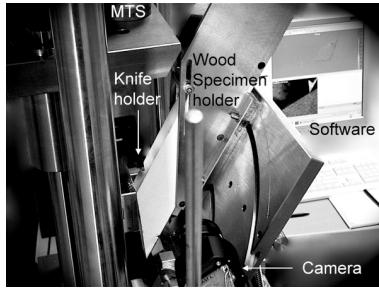
In the state of the art, two kinds of experiments can be performed to study the chip size distribution. The first one consists in using industrial slabbers in real conditions with green wood logs at high cutting speed ($V_c \simeq 3600$ m/min). An example of this kind of experiment is shown in Figure 1a. The parameters to be tested must include cutting kinematics, tool geometry, log positioning and log parameters. Chips produced are then screened and their size distribution carefully observed (Hernandez and Boulanger, 1997; Hernandez and Quirion, 1993; Laganière, 2004).

The second approach deals with mesoscopic characteristics. Chipping experiments on wood specimens are carried out with a compression testing machine with a

specific experimental setup (Fig. 1b). For these experiments in quasi static conditions ($V_c \simeq 60$ mm/min), close attention is paid to the area near the cutting edge. Tested parameters are unchanged with respect to the previous macroscopic process. Chip size distribution, cutting forces, failure modes, strain and stress fields are analysed in order to understand chipping phenomena (Buchanan and Duchnicki, 1963; Hellström, 2008b; Twaddle, 1997; Uhmeier, 1995).



(a) Chipping experiment with a slabber head (Forintek Canada Corp, Quebec)



(b) Chipping experiment with compression testing machine (Hellström, 2008b)

Fig. 1: Chipping experimental setups

So as to offer a better understanding of the physical chipping processes, this paper focuses on two approaches of chip production. The first one refers to the observation of fragmentation phenomena with a specific experimental setup in dynamic conditions ($V_c = 400$ m/min), which allows the extraction of influent parameters like grain direction and nominal cut thickness.

The second one consists of a geometrical and kinematic study of the milling process in order to analyse the effect of cutting geometrical and kinematic parameters on the local cutting conditions. This study will give guidelines to milling machine manufacturers to help them to improve machine design and optimize chips formation.

In the whole paper, except for grain direction, all geometric and kinematic abbreviations are taken according to ISO 3002 standard (ISO, 1982, 1984).

2 Experimental analysis in orthogonal cutting

2.1 Local grain direction definition

Although the experimental work is carried out in orthogonal cutting conditions, it is necessary to define locally the grain direction in complex 3D cases. This notion was tackled by Kivimaa (1950) and McKenzie (1960) with the cutting modes defined by two angles. However these two angles were not well defined for complex 3D cases. In Figure 2, a new definition is proposed. Locally, the grain direction is defined with two angles: the Grain Edge Inclination angle GEI and the Grain Direction angle GD . The GEI angle is measured in the grain cutting edge plane P_{sGD} , containing the cutting edge and the grain direction. This angle is chosen to vary between 0 and 90°. The GD angle is measured in the grain working plane P_{fGD} , containing the resultant cutting and the grain directions. The resultant cutting direction contains the cutting direction, as well as the feed direction (ISO, 1982). GD is an oriented angle from the resulting cutting edge direction and the grain direction oriented towards the cutting face. So GD varies between 0 and 180°. In Figure 2 appears the working cutting edge plane P_{se} defined in (ISO, 1982).

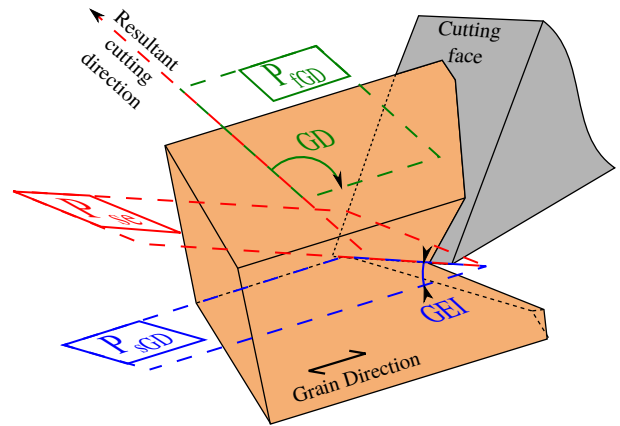


Fig. 2: Definitions of the Grain Edge Inclination angle GEI and the Grain Direction angle GD . GEI is measured in the grain cutting edge plane P_{sGD} , containing the cutting edge and the grain direction. GD is measured in the grain working plane P_{fGD} , containing the resultant cutting and the grain directions. P_{se} is the working cutting edge plane, defined in (ISO, 1982).

2.2 Experimental method

Chipping observations are carried out on a Chardin's pendulum, similar to a Charpy's pendulum, but designed to measure cutting forces in sawing (Chardin,

1958). It was used to obtain instrumented orthogonal cutting experiments to characterize cutting forces on dry wood (Eyma et al, 2005). The arm of the pendulum is 1.2 m long and weighs 36 kg. So the available kinematic energy reaches 756 J and the cutting speed V_c 400 m/min.

Contrary to its usual practice, the specimen holder is placed at the arm extremity. The specimen holder is 140 mm long and is equal to the specimen and the cutting lengths. A tooth coming from an industrial milling cutter is used for these experiments. The normal rake angle is set to $\gamma_n = 45^\circ$, the normal clearance angle to $\alpha_n = 5^\circ$. The tool holder, in green in Figure 3, is attached to a Kistler piezo-electric force transducer (maximum allowable effort $F_{max} = 10$ kN) to measure cutting forces. The cutting process is observed via a Phantom v9.1 camera (maximum resolution 1632×1200 pixels at 1000 fps (frames per second), exposure time: $6 \mu s$).

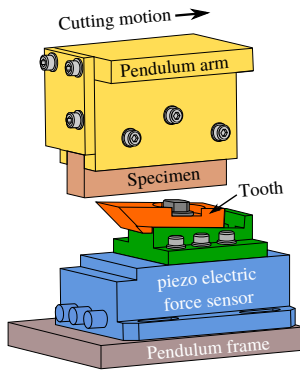


Fig. 3: Chardin's pendulum



Fig. 4: Preliminary results (Beech, nominal cut thickness $h_D = 20$ mm, nominal cut width $b_D = 20$ mm)

Average industrial milling conditions usually take a feed per tooth f_z of more or less 30 mm and a back engagement of the cutting edge (or named cutting depth) a_p of more or less 50 mm which represent an average section of 1500 mm^2 . Here, our preliminary experiments show that the pendulum is unable to cut a 400 mm^2

green beech section on a 140 mm cutting length. Figure 4 shows an uncompleted cut of a 400 mm^2 section of beech and the definitions of the nominal cut thickness h_D and the nominal cut width b_D . As a consequence, an experimental design is set and uses the following parameters: nominal cut thicknesses $h_D = 6, 10, 14$ mm, nominal cut widths $b_D = 6, 10, 14$ mm and Grain Direction angles $GD = 70, 90, 110^\circ$. The GEI angle is kept at 90° . Two wood species are selected: beech for its homogeneity and Douglas fir known for its heterogeneity and bad machinability. The Moisture Content (MC) is controlled after the experiments by a double weight technique (Eq 1) (ASTM, 1994). Each set of parameters is repeated 5 times. Specimens are machined from fresh beech and Douglas fir slabs, wrapped in plastic films and put into a fridge to limit MC decrease.

$$MC = \frac{M_{green} - M_{dry}}{M_{dry}} \times 100 \quad (1)$$

2.3 Results and discussion

The first cutting mechanism observed shows a chip fragmentation for beech and Douglas fir when $GD \leq 90^\circ$. It is named Mechanism 1 on the Figures 5a and 5b. These chipping phenomena summed up in Figure 6 agree with previous observations published by McKenzie (1960). First of all, the cutting edge bends wood fibers (Fig. 6a). This bending is due to cell wall buckling. Then, when wood cells finish collapsing, fibers get sliced (Fig. 6b). Because of the cutting motion, sliced fibers are longitudinally compressed under the cutting face. When compression stress is higher than shear strength, shearing between fibers occurs, which ejects free water from the cells (Fig. 6c). This water ejection occurs only when MC is higher than 35%.

These phenomena are comparable to the splitting mechanisms observed for green wood peeling. As soon as the shear strength of green wood is exceeded, lathe checks appear (Thibaut and Beauchêne, 2004). For homogeneous species like beech, checks are almost periodic (Denaud et al, 2012; Palubicki et al, 2010).

Although Hernandez and Quirion (1993) studied the influence of the chip breaker (commonly named counter knife in wood industry) on chipping, this apparatus is useless at this scale of nominal cutting thickness ($h_D \leq 14$ mm) because chips are fully fragmented on the tool face.

As it could be observed in quasi-static cutting tests (Buchanan and Duchnicki, 1963; Hellström, 2008b; Uhmeier, 1995), the average fragmented chip thickness C_t

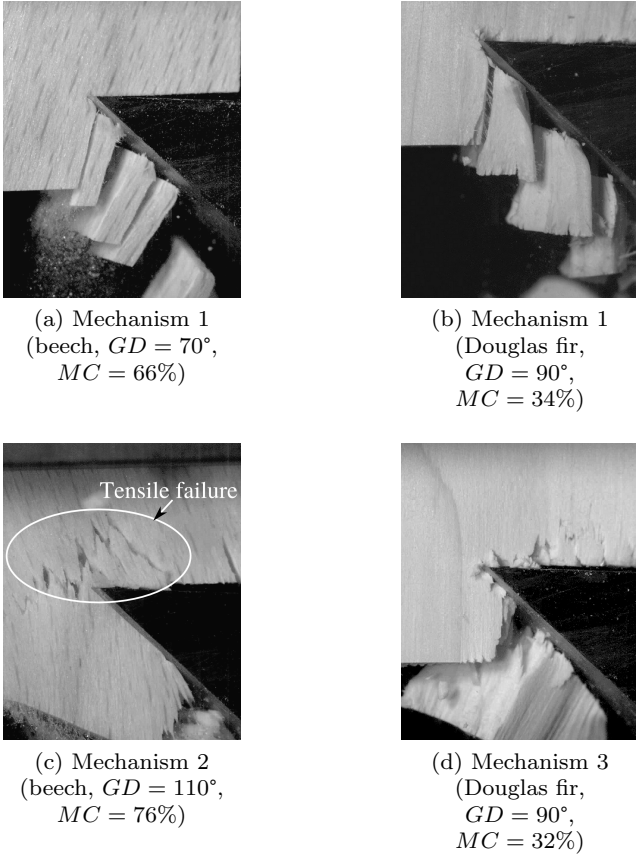


Fig. 5: Observation of different chip formation mechanisms ($V_c = 400$ m/min, $b_D = 10$ mm, $h_D = 10$ mm)

(Fig. 6a), obtained with Chardin's pendulum, is proportional to the nominal cutting thickness h_D and independent of the nominal cutting width b_D . Figure 7 shows the evolution of the mean chip thickness functions of the nominal cutting thickness for beech and Douglas fir.

As observed by Twaddle (1997), chip thickness is inversely proportional to the specific density. So with the same experimental parameters, Douglas fir chips remain thicker than beech chips. Mechanical properties of green beech and Douglas fir are summed up in Table 1.

However contrary to the observation from Buchanan and Duchnicki (1963) and Twaddle (1997), chip thickness is not proportional here to the ratio shear strength on crushing strength. Moreover, the slope of the curves in Figure 7 does not correspond to Muorto model (Buchanan and Duchnicki, 1963). Many reasons can explain these differences. First, dynamic mechanical properties of green beech and Douglas fir may be really far from those obtained by Bodig and Jayne (1982). Second, the ANOVA model developed by Twaddle (1997)

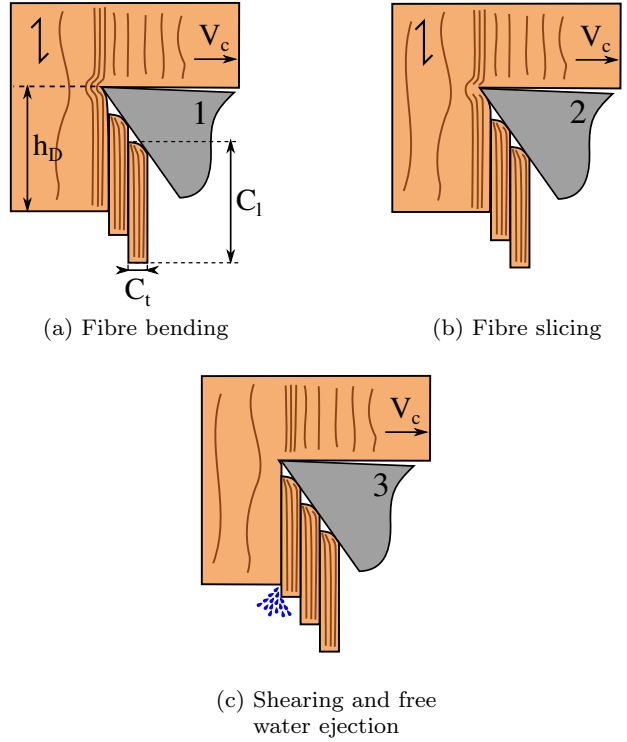


Fig. 6: Chip fragmentation mechanism

| Species | Specific density | Crushing strength (MPa) | Shear strength (MPa) | $\frac{Shear}{Crushing}$ |
|-------------|------------------|-------------------------|----------------------|--------------------------|
| beech | 0.57 | 11.9 | 5.7 | 0.47 |
| Douglas fir | 0.45 | 13.7 | 5.0 | 0.36 |

Table 1: Mean longitudinal properties of green beech and Douglas fir (Bodig and Jayne, 1982)

may not be extend to other species or to higher cutting speeds. Last but not least, although specimens were machined from the same slabs at the same time and experiments carried out in consecutive days, large variations on MC appeared (20 to 60% for Douglas fir and 30 to 90% for beech). They can be explained by the fast drying of the thinnest wood specimens (a couple of hours). However the Moisture Content influences the chip thickness (Buchanan and Duchnicki, 1963). Consequently to these material variations, additional studies should be carried out in order to decorrelate the effect of specific density, mechanical properties and moisture content in order to propose a phenomenological model of fragmented chip thickness.

Chip formation, directly observed thanks to the camera at 1000 fps, shows that fragmented chip thickness decreases slowly during the cut. Two factors can explain this variation. First, the motion of Chardin's pendulum induces a GD decrease of 7° during the cut.

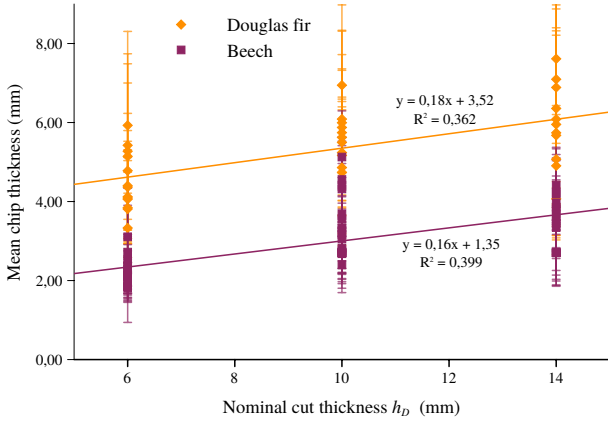


Fig. 7: Mean chip thickness with standard deviation depending on cut thickness h_D (beech and Douglas fir, $GD = 90^\circ$)

Wood mechanical characteristics depend on GD , because it affects shearing strength which decreases when $GD \neq 90^\circ$ (Goli et al, 2010) and induces thinner chip production. Then, as the pendulum falls freely, V_c decreases of about 20% during the cut. The effect of the loss of speed must be analysed separately. Nevertheless, cutting speed cannot remain constant with our equipment.

Average cutting forces measurements are carried out with piezoelectric force transducers. In Figure 8 the obtained signal is close to the results published by Hellström et al (2011) with the appearance of regular peaks and much noise. The peaks measured which could be related to chip formation, have a period of about 0.4 ms. This corresponds to a phenomenon, that arises every 2.8 mm at a cutting speed of 7 m.s^{-1} , compatible with chip formation (see Figure 7). However, noise with a period near 0.07 ms (so a frequency of 14 kHz) highly perturbs the measurement. Moreover, here, the natural frequency of the force transducers is about 4 kHz, decreasing with the mass of the tool holder and the tool. The chipping frequency (between 1 and 4 kHz), determined with the mean chip thickness and the cutting speed, can be higher than the cutting system natural frequency. At least, the quality of the tool holder attachment (surface flatness, applied torque, ...) can perturb the force measurement via parasite vibrations. As a consequence, we cannot confirm the force peak before chip fragmentation as previously shown by Uhmeier (1995) and McKenzie (1960) in quasi-static conditions. A solution would consist in filming the cut with additional frames per second and in synchronising the movie with the force signals.

When the grain direction is higher than 90° , big wood particles are teared by tension stress on fibers (Fig. 5c). Within this second mechanism, chip wood

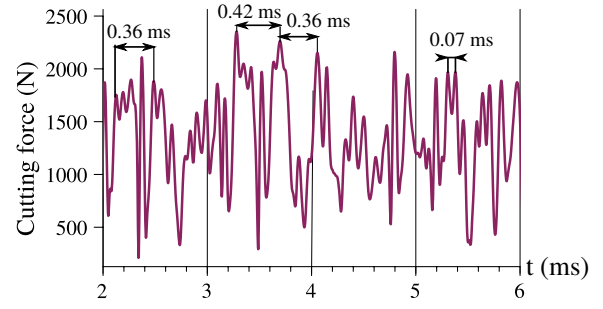


Fig. 8: Cutting force as a function of time (beech, $h_D = 10 \text{ mm}$, $b_D = 10 \text{ mm}$, $GD = 90^\circ$)

cells are also compressed and free water is partially ejected. This ejection is mostly observed with beech probably because of the size of its vessels.

A third cutting mechanism is found: shearing between growing rings on Douglas fir (Fig. 5d). It appears only when the specimen contains only few growing rings in the specimen thickness. More specific experiments must be performed to determine the limit ratio h_D/b_D needed for chip production.

The second and third cutting mechanisms are not appropriate to give chips with standard dimensions as required by pulping industry. The milling process parameters have to be chosen to improve chip formation with the first mechanism.

3 Geometric and kinematic modelling

3.1 Method

3.1.1 Slabber head geometry

A slabber head as represented in Figure 9 consists of a conical milling cutter composed of fragmentation teeth and a saw blade for finishing. In order to design these slabber heads, each industrial needs to define geometrically the active parts. In the literature, the most complete head definition was found in Forintek Canada Corporation (Laganière, 2006). However, this definition does not agree with the ISO 3002 standard (ISO, 1982) in use in machining area. A complete geometric definition is proposed based on a CT700 slabber head provided by LBL-BRENTA-CD (Fig. 9). Only the chipping teeth are studied throughout this paper. The conical milling cutter is composed of 3 rows with 3 teeth on each. The different geometrical parameters are gathered in Table 2. This head is the support for the kinematic study in the next section.

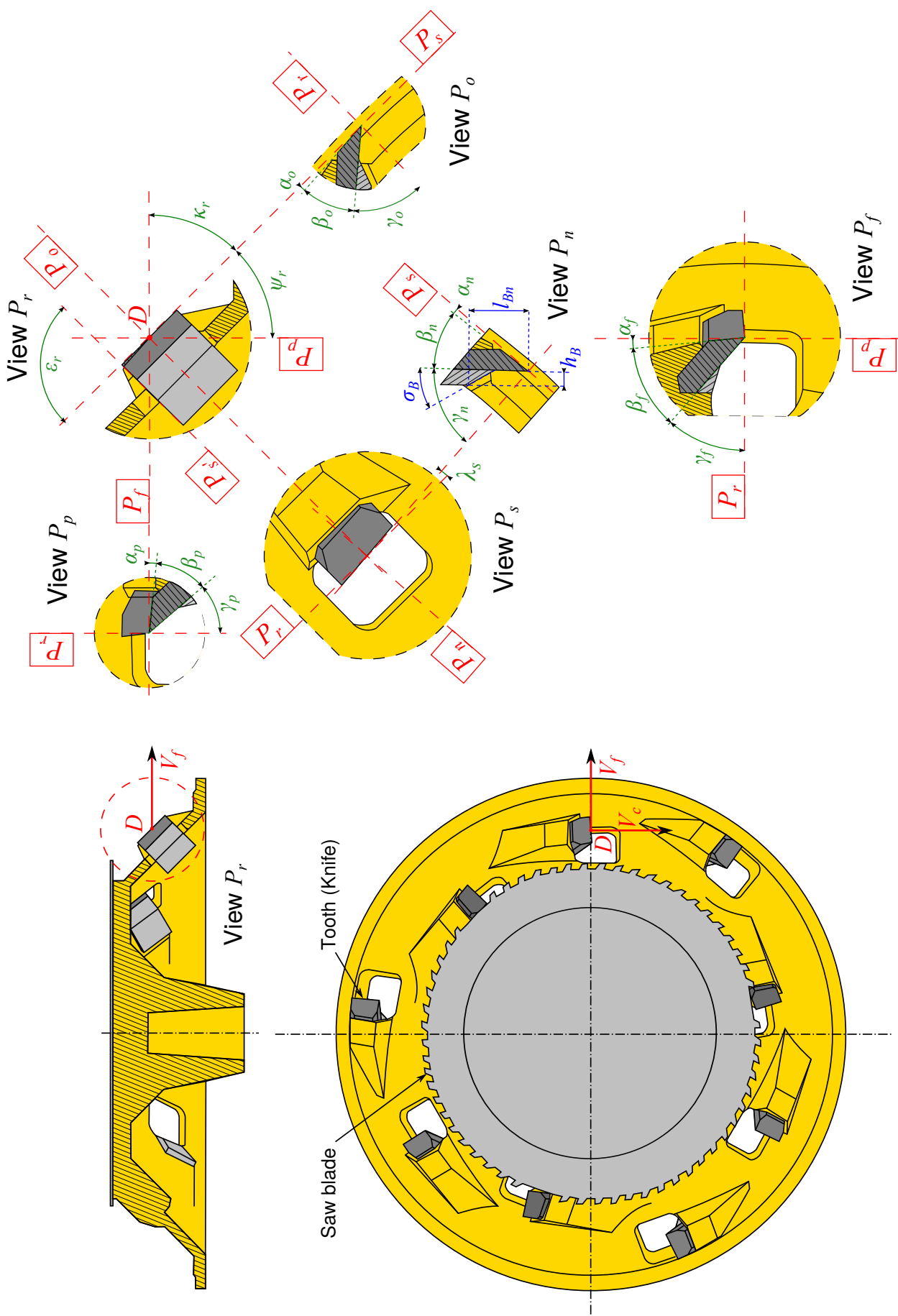


Fig. 9: Tool angles definition in tool-in-hand system for a slabber head with split knives (3 teeth on 3 rows) according to (ISO, 1982).

| Symbol | Name | Symbol | Name |
|------------|------------------------------|--------------|---------------------------------|
| D | Cutting edge principal point | κ_r | Tool cutting edge angle |
| P_r | Tool reference plane | ψ_r | Tool lead angle |
| P_f | Assumed working plane | λ_s | Tool cutting edge inclination |
| P_p | Tool back plane | ϵ_r | Tool included angle |
| P_s | Tool cutting edge plane | γ_n | Tool normal rake angle |
| P_n | Cutting edge normal plane | γ_f | Tool side rake angle |
| P_o | Tool orthogonal plane | γ_p | Tool back rake angle |
| l_{B_n} | Chip breaker distance | γ_o | Tool orthogonal rake |
| h_B | Chip breaker height | β_n | Normal wedge angle |
| σ_B | Chip breaker wedge angle | β_f | Tool side wedge angle |
| α_n | Tool normal clearance angle | β_p | Tool back wedge angle |
| α_f | Tool side clearance angle | β_o | Tool orthogonal wedge angle |
| α_p | Tool back clearance angle | α_o | Tool orthogonal clearance angle |

Table 2: Notations used in Fig.9 (From ISO 3002-1 standard (ISO, 1982))

3.1.2 Kinematic modelling

The kinematic study allows computing effective cutting edges paths and cutting area, necessary to cutting forces models (Kaymakci et al, 2012) and (Mckenzie et al, 2001). The previous geometrical definition is used to develop a generic and parameterizable slabber head on Mathematica[®] software. The Figure 10 shows the generic slabber used. The angular position of the head is defined by the parameter θ . The following parameters are introduced:

- **Kinematic:** Cutting speed V_c , feed f_z and $Cant$.
- **Log geometry:** Minimal log radius R_{LogMin} , log tape $LogTape$, vertical distance between the carriage and the slabber head axis $H_{Carriage}$.
- **Slabber head geometry:** Head diameter $R_{HeadMin}$, number of teeth Z_{mn} (with m the number of teeth

per rows and m number of rows), tooth length l_s , tool cutting edge angle κ_r , tool cutting edge inclination λ_s , normal rake angle γ_n , normal wedge angle β_n , normal clearance angle α_n .

The proposed model computes working angles, grain direction GD and nominal cross-section area of cut A_D , which is a function of the position of the head, while taking into account cutting kinematic motion. We use the methods developed by Fromentin and Poulachon (2010a,b).

As Mathematica[®] is a symbolic calculation software, parametric curves are used to model the log surface (Equation 2) and the cutting edges paths Z_{mn} for tooth m row n (Equation 3). Here cutting and feed motions are applied to the slabber head and the log stands still.

$$LogSurface[x, \psi] = \begin{cases} x \\ (R_{LogMin} - x * LogTape) * \sin(\psi) + CarriageShift + R_{LogMin} \\ (R_{LogMin} - x * LogTape) * \cos(\psi) - Cant/2 \end{cases} \quad (2)$$

$$PathZ_{mn}[t, z] = \begin{cases} (R_{Headmin} + z * \tan \kappa_R) * \cos \left(\frac{V_c * t}{R_{Headmin}} - (m-1) \frac{2\pi}{N_{knife}} + (n-1) \frac{2\pi * N_{rank}}{N_{knife}} \right) \\ + z * \frac{\tan \lambda_S}{\cos \kappa_R} * \sin \left(\frac{V_c * t}{R_{Headmin}} \right) + \frac{V_c * f_z * N_{knife} * t}{R_{Headmin}} \\ (R_{Headmin} + z * \tan \kappa_R) * \sin \left(\frac{V_c * t}{R_{Headmin}} - (m-1) \frac{2\pi}{N_{knife}} + (n-1) \frac{2\pi * N_{rank}}{N_{knife}} \right) \\ - z * \frac{\tan \lambda_S}{\cos \kappa_R} * \cos \left(\frac{V_c * t}{R_{Headmin}} \right) \\ z \end{cases} \quad (3)$$

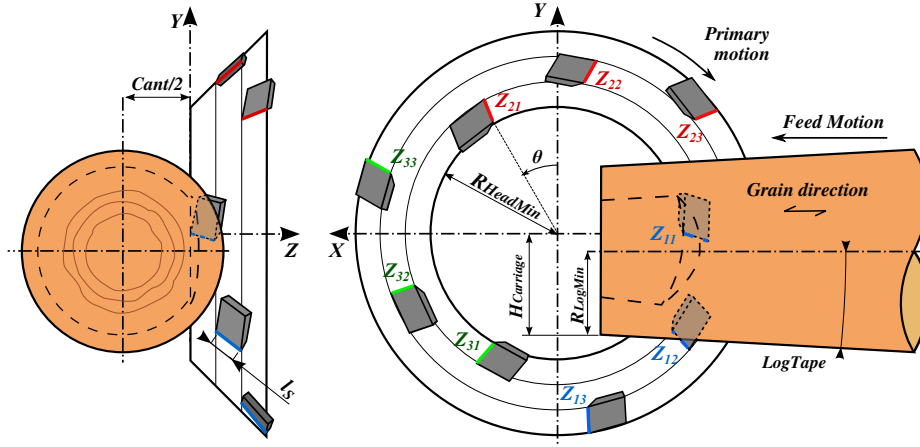
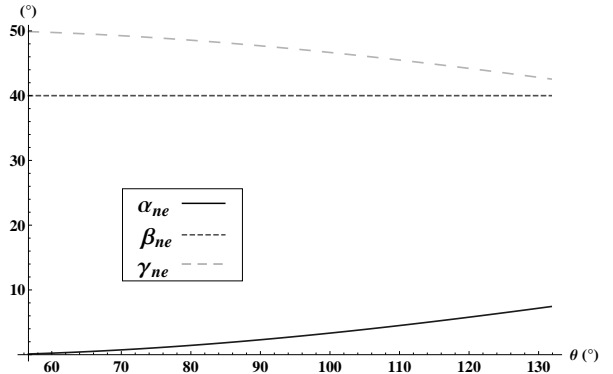


Fig. 10: Parameters used in the model (Z_{mn} : tooth m row n).

From these equations, the evolution of the Grain Direction angle along the cut can be followed. The evolutions of the working angles and the nominal cross-sectional area of cut are computed according to ISO 3002 standard (ISO, 1982).

An example obtained with this geometrical approach is given in section 3.2. The following values are implemented into the program (Config 1): $V_c = 3600$ m/min, $f_z = 35$ mm, $Cant = 400$ mm, $R_{LogMin} = 300$ mm, $LogTape = 1\%$, $H_{Carriage} = 320$ mm, $R_{Headmin} = 320$ mm, $l_S = 60$ mm, $\kappa_R = 45^\circ$, $\lambda_S = 0^\circ$, $\gamma_n = 45^\circ$, $\beta_n = 40^\circ$, $\alpha_n = 5^\circ$. Here the slabber head contains 3 teeth per row on 3 rows. For this configuration, as the grain direction is equal to the feed direction and as the tool cutting edge inclination is equal to zero, the Grain Edge Inclination angle GEI is equal to the tool cutting edge angle κ_r .

γ_{ne} decreases regularly from 50° at tooth entrance to 42.5° when tooth exits. α_{ne} increases from 0° to 7.5° .



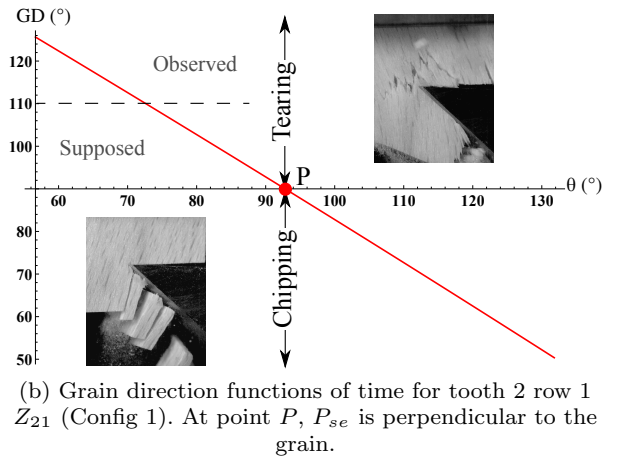
(a) Working angles for tooth 2 row 1 Z_{21} measured in plane P_{ne} (Config 1)

3.2 Results and discussion

3.2.1 Working angles

Working angles are defined in the tool-in-use system taking into account the action of the tool in the cutting process (ISO, 1982). Figure 11a shows the evolution of the working normal rake angle γ_{ne} , the working normal clearance angle α_{ne} . By definition the working normal bevel angle β_{ne} remains constant. Here, due to the high feed ($f_z \simeq 30$ mm), a particular attention is paid to the working normal clearance angle α_{ne} . If α_{ne} becomes negative, ploughing (penetration of the tool flank into the cut surface) occurs.

Since mills with small diameters are more sensitive to ploughing, working angles on tooth 2 row 1 (Z_{21}) are analysed. With the following parameters in parenthesis ($\gamma_n = 45^\circ$ and $\alpha_n = 5^\circ$), it is observed that



(b) Grain direction functions of time for tooth 2 row 1 Z_{21} (Config 1). At point P , P_{se} is perpendicular to the grain.

Fig. 11: Milling model outputs

Thus, with the large feed chosen ($f_z = 35$ mm) ploughing can be observed at tooth entrance in wood

($\alpha_{ne} \leq 0^\circ$). Several simulation results are presented in Table 3, evaluating parameters effects. In order to avoid ploughing, α_{ne} and $H_{Carriage}$ must be increased (Config 2 in Table 3) and f_z diminished (Config 3 in Table 3). At fixed $R_{HeadMin}$, the variation range of working angles grows when R_{LogMin} increases and shrinks with $Cant$.

Industrially, even though changing α_{ne} requires a tooth substitution, modifying $H_{Carriage}$ implies to reconsider all the slabber process installation, the machine being composed only of x and z axis (Fig. 10).

| Config | $R_{HeadMin}$ (mm) | Z | $H_{Carriage}$ (mm) | f_z (mm) |
|--------|----------------------------|---------------------|---|---------------|
| 1 | 320 | 3 | 320 | 35 |
| 2 | 320 | 3 | 220 | 35 |
| 3 | 320 | 3 | 320 | 25 |
| 4 | 640 | 3 | 320 | 35 |
| 5 | 640 | 6 | 320 | 35 |
| Config | α_{ne} ($^\circ$) | GD ($^\circ$) | Discontinuity $Z_{21} - Z_{33}$ ($^\circ$) | |
| 1 | 0 - 7.5 | 125.6 - 50.4 | -24.0 | |
| 2 | -0.5 - 3.8 | 148.5 - 70.5 | -15.2 | |
| 3 | 1.0 - 7.2 | 125 - 49.8 | -23.1 | |
| 4 | 2.8 - 5.3 | 109.7 - 72.1 | 9.1 | |
| 5 | 1.4 - 4.9 | 109.7 - 72.1 | -11.1 | |

Table 3: Effect of $R_{HeadMin}$, Z , $H_{Carriage}$ and f_z on α_{ne} , GD and the cutting continuity. Others parameters remain constants. Bold values are undesirable industrial operating points.

It is well known that ploughing highly increases flank wear and may affect chip formation mechanisms by generating longitudinal fractures. When ploughing occurs, the extremity of wood cells is damaged. Recent works (Hellström, 2010) in pulping process shows that wood axial precompression treatment is the most efficient method to reduce refining energy for thermomechanical process. In fact precompression of wood chips opens the wood structure, which increases the specific surface and facilitates the penetration and uptake of chemicals and water. For chemical refining process, Buchanan and Duchnicki (1963) show that damaged cells can induce heterogeneous impregnations of chemicals in wood chips, which reduces paper quality.

Some studies (Buchanan and Duchnicki, 1963; Hellström, 2008a) showed the influence of β_n on fragmentation mode. When β_n increases, fracturation occurs in mode II (shearing) instead of mode I (opening). The transition between these modes is around 40° . Mode I is less energy consuming than mode II (Uhmeier and Persson, 1997). Moreover mode II is more cell damaging than mode I (Buchanan and Duchnicki, 1963). So

wood tool providers should produce teeth with a maximal bevel angle of 40° .

3.2.2 Grain direction

The grain direction angle GD , defined in Figure 2, can vary from 0 to 180° . In the present application, in Figure 11b, GD decreases quasi-linearly during the cut. In the first configuration (Table 3), it starts at 125.6° , when point D of Z_{21} enters into the log, to finish at 50.4° when Z_{21} exits. Point P represents the point where the P_{se} plane is perpendicular to the grain, which induces $GD = 90^\circ$. When GD is smaller than 90° , side grain cutting occurs. In these conditions, the grain is globally perpendicular to face A_γ which makes fragmentation easier (Fig.5a). However when GD is higher than 90° , cross grain cutting can occur (observed in experiments when $GD \geq 110^\circ$). Globally the grain is parallel to the face. Chips are not split but cutting occurs by tear of large parts of the wood as in the Figure 5c.

In order to favour chip fragmentation, the tooth must penetrate the log when GD is smaller than 90° that is when the cutting edge is under the slabber head axis. $H_{Carriage}$ must be as high as possible (Config 2 in Table 3). Moreover Laganière (2004) shows that when $H_{Carriage}$ is low, more fines appear, which decreases chip quality. The range of the GD angle augments with the R_{LogMin} and diminishes with $Cant$. The ideal configuration would be $H_{Carriage} \geq 2.R_{LogMin}$, which would represent a pure down milling situation. Thus, all these variables affect the machine tool design.

3.2.3 Nominal cross-sectional area of cut

The nominal cross-sectional area of cut A_D (ISO, 1984) allows to calculate cutting forces F_c thanks to a simple cutting law $F_c = A_D \times K_c$ where K_c is the specific cutting force. This study is only geometric and kinematic so only A_D is taken into account. Moreover K_c depends on many parameters like GD and has not been defined for green wood yet. Usually, the following simplification is currently done: $A_D = b_D \times h_D$, but here exact nominal cross-sectional area of cut is computed.

By analogy with the work of (Fromentin and Poulachon, 2010b), the first step consists in determining the nominal width of cut b_D . b_D is defined as the portion of cutting edge in contact with the log and is equivalent to the fragmented chip width C_w (Fig. 12). In Figure 13a for teeth Z_{21} , Z_{22} , Z_{31} and Z_{32} , the nominal width of cut increases quasi-linearly at tooth entrance until the tool length l_S is reached (here $l_S = 60$ mm) and decreases at log exit. Although $\lambda_s = 0^\circ$, b_D augments slowly because of the log geometry.

For chosen parameters (Config 1 in Table 3), cut discontinuity does not occur between teeth Z_{m1} exits and teeth $Z_{(m+1)3}$ entrances (Fig. 13a). However it clearly appears on configuration 4 in Table 3. Industrially, this should generate chocks and vibrations in the slabber head. To avoid this problem, the number of teeth per row, the number of rows, the values of R_{LogMin} or $H_{Carriage}$ can be increased (Config 5 in Table 3). $R_{HeadMin}$ can be decreased too but only if small diameter logs are surfaced. In order to diminish cutting forces, the tool length l_s must be reduced.

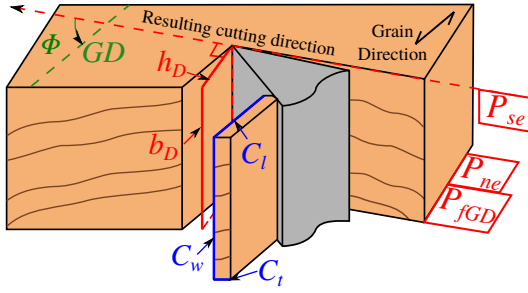
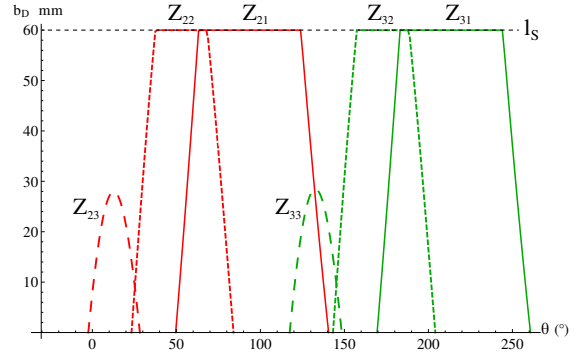


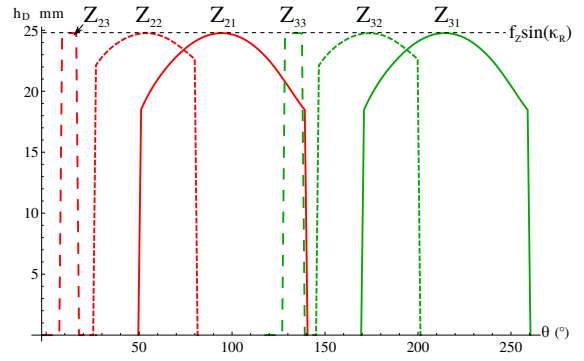
Fig. 12: Definition of b_D , h_D , C_l , C_w and C_t in the case of surfacing with slabber head. Here the planes P_{ne} and P_{fGD} , defined in Figure 2, are equal.

Chips are always produced in the grain direction. Moreover for homogeneous green wood, the plane P_{sGD} is the fragmentation plane. A defined by Merchant (1945), this plane can also be called the shear plane. The equality between the shear angle Φ , mostly used in machining (Merchant, 1945), and the Grain Direction angle GD makes that the nominal thickness of cut h_D is not equal to the fragmented chip length C_l (Fig. 12). Chips split in the wood grain direction, here along the x axis. So the fragmented chip length C_l is in fact equal to the feed f_z . h_D represents the distance perpendicular to the tooth path between the tooth path and the previous one. In Figure 13b, for rows 1 and 2, the nominal chip thickness increases up to $h_{DMax} = f_z \sin(\kappa_r)$, corresponding to the point where plane P_{se} is perpendicular to feed motion (x axis), and then decreases to 0. To reduce h_D , that is to diminish the cutting forces, the cutting edge must penetrate into the log only under the slabber head axis. $R_{HeadMin}$ must be increased and $H_{Carriage}$ too.

The nominal cross-section area of cut A_D is defined as the area normal to the cut surface up to the log free surface. A_D is a quadrangle, its area is exactly computed (Fig. 14). Geometrical results are validated with a geometrical model on a CAD software. Differences between these two models are less than 5% and are due



(a) Nominal width of cut b_D for teeth 2 and 3 rows 1 to 3. (Config 1)



(b) Nominal thickness of cut h_D for teeth 2 and 3 rows 1 to 3. (Config 1)

Fig. 13: Nominal cutting dimensions

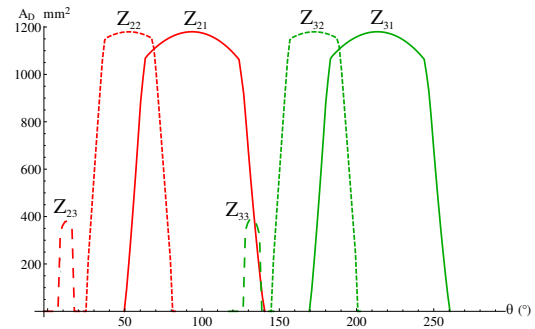


Fig. 14: Nominal cross-sectional area of cut A_D for teeth 2 and 3 rows 1 to 3. (Config 1)

to circular tooth path on the CAD software instead of cycloidal in the kinematic model.

Although Lusth et al (2012) start to work on energy consumption for slabber heads, no specific cutting force law K_c has been defined for green wood yet, concluding about cutting forces is impossible for the moment.

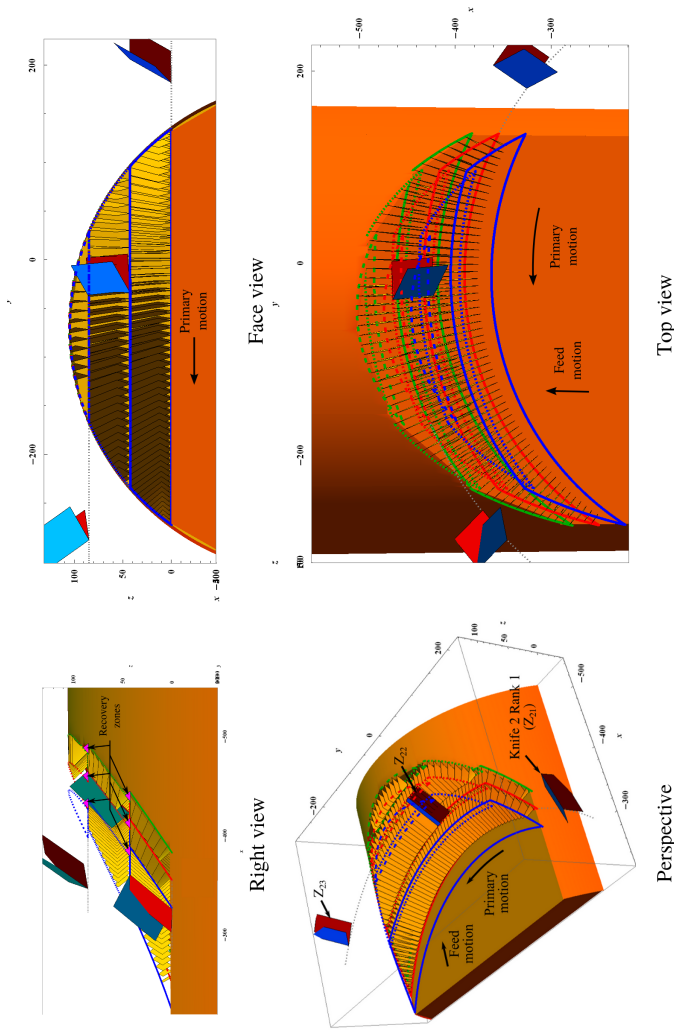


Fig. 15: View from kinematic animation showing the log, three knives and their paths into the log (Z_{21} : solid line. Z_{22} : thin dotted. Z_{23} : wide dotted.). Some cross-sectional area of the cut are also represented.

3.2.4 Kinematic animation

In order to improve the cutting kinematic understanding, an animation is generated. The extract (Fig. 15) shows the log, the teeth and their paths and the nominal cross-sectional area of cut at different time steps. Cutting kinematic can be summed up this way. Each tooth cuts the log from the outside to the inside. The order for this example is the following: $Z_{13} \rightarrow Z_{12} \rightarrow Z_{11} \rightarrow Z_{23} \rightarrow Z_{22} \rightarrow Z_{21} \rightarrow Z_{33} \rightarrow Z_{32} \rightarrow Z_{31}$ (Fig.10). The surface generated by 3 rows of teeth includes steps, which are due to feed rates. To avoid the production of fine chips in the teeth recovery zones, the teeth should have a minor cutting edge angle κ'_r (ISO, 1982) at least equal to the cutting edge angle κ_r .

4 Conclusion and perspectives

Pulp industry needs to control chips size distribution to get an efficient production. To provide a better understanding of the green wood chipping production in dynamic conditions, two axes were explored.

A Chardin's pendulum was first used to study chip formation mechanisms in order to reproduce quasi-static observations. Chipping phenomena were obtained with $GD \leq 90^\circ$. High framed rate cutting films allow the observation of the ejection of free water during chip fragmentation. At our nominal chip thickness ($h_D \leq 14$ mm) it was demonstrated that chip breaker was useless. Other cutting phenomena were established like shearing between growing circles.

To complete this work on fundamental cutting aspects, other parameters should be studied like tool angles or cutting speed. Some modifications on the experimental setup will be done in order to increase the available energy, avoid the cutting speed decrease during the cut and measure the variation of the cutting speed.

A geometric and kinematic modelling of rough milling process at high feed was carried out so as to improve machine design and optimize cutting geometry. The model showed that increasing $H_{Carriage}$ avoided ploughing and augmented good chipping zone in relation with the milling cutter geometry. Numerous teeth are needed to observe good cutting continuity.

To go further, the effect of the saw blade and the secondary cutting edge should be studied. Research on cutting law for green wood depending one GD and H should be performed to establish mechanical model of cutting forces.

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